NFPA 22

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Proposal: Delete the term "Standpipe" as it is not consistently used throughout the standard and is redundant with the term suction or gravity tank.

14.2.5.1.1 The base elbow of tanks with steel-plate tank risers, of suction tanks, or of ~~standpipes~~ gravity tanks shall have bell ends.

14.2.8.2.3 A rigid flanged connection or welded joint shall be permitted to be used between the discharge pipe and the bottom of a suction tank, a ~~standpipe~~ gravity tank , or the base of a steel-plate tank riser of a tank that is located on an independent tower where special approval is obtained from the authority having jurisdiction.

16.2.1.1 A steam-heated vertical radiator system shall be used for elevated tanks with unprotected tank risers of 3 ft (0.91 m) or more in diameter that have tower heights under 100 ft (30.5 m) *(see 13.1.2)*, ~~standpipes~~ gravity tanks, and on-grade suction tanks.

Already proposed to be deleted see Chapter 16 proposal.

16.2.1.3 Immersed steam coils shall be used for suction tanks and ~~standpipes~~ gravity tanks that have flat bottoms supported near ground level in situations where the tank is kept filled so that the steam coils are continuously submerged.

Already proposed to be deleted see Chapter 16 proposal.

16.2.2.1 Suction tanks, ~~standpipes~~, and ~~elevated~~ gravity tanks that have tank risers of 3 ft (0.91 m) or more in diameter shall not be required to have provision for heat.

Already proposed to be deleted see Chapter 16 proposal.

16.3.6.2 An accurate angle socket thermometer that has at least a 6 in. (152 mm) stem and that is calibrated as low as 30°F(−1.1°C) shall be permanently inserted through the plate ~~or standpipe~~ and as far from the heating unit as possible.

16.3.8.9 The surface water temperatures for elevated gravity tanks, ~~standpipes~~, and suction tanks shall be ascertained by means of a listed temperature-detecting device.

A.16.1.3 Choice of Circulating Heaters. To select a suitable circulating heater, first obtain from Figure 16.1.4 the lowest mean atmospheric temperature for one day that may occur at the locality in question; then determine the total heat loss from the

tank equipment in British thermal units (kilowatts) per hour from Table 16.1.4(a) for an ~~elevated~~ gravity steel tank, from Table16.1.4(b) for an elevated wood tank, from Table 16.1.4(c) for a steel suction tank ~~or standpipe~~, or from Table 16.1.4(d) for an embankment-supported coated fabric suction tank. The heater installed should have sufficient capacity to deliver, under actual field conditions, an amount of heat that is equivalent to that lost from the tank equipment. A steam water heater should be planned with due consideration of the steam pressure available. Other heaters should be planned for the particular kind of fuel to be used.

A.16.1.4 Heat Losses. Table 16.1.4(a) through Table 16.1.4(h)specify the heat losses from uninsulated ~~elevated~~ gravity steel tanks, elevated wood tanks, steel suction tanks ~~and standpipes~~, embankment-supported coated fabric suction tanks, insulated steel gravity tanks, and insulated steel suction tanks, respectively, for common sizes exposed to various atmospheric temperatures of 35°F to −60°F (1.7°C to −51.1°C). The losses are indicated in the British thermal units per hour (kilowatts) that are lost from the entire tank equipment when the temperature of the coldest water is safely above the freezing point, and represent the British thermal units per hour that the heating system should supply when the atmospheric temperature is within the range provided by the tables.

A.16.2 Recommendations for Gravity Circulation Heating. Gravity circulation allows convenient observation of the coldest water temperatures at a thermometer in the cold-water return pipe and is dependable and economical when correctly planned. Cold water received through a